

2.10: Non-dimensional form of Navier-Stokes equation (ρ constant):

Navier-stokes equation is in general a dimensional equation.

Dimension of every term is $\frac{\rho U^2}{L}$, where U is some typical (representative) velocity, L is some typical length and ρ is constant density.

Let us take the equation of motion in cartesian coordinates along x -axis. It is

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right) \rightarrow (1)$$

Let us introduce

$$u' = \frac{u}{U}, \quad v' = \frac{v}{U}, \quad w' = \frac{w}{U}$$

$$x' = \frac{x}{L}, \quad y' = \frac{y}{L}, \quad z' = \frac{z}{L}, \quad t' = \frac{U}{L} t, \quad p' = \frac{p}{\rho U^2}$$

where $u', v', w', x', y', z', t', p'$ represents non-dimensional velocity component, time and pressure etc. in the equation (1), then we have,

$$\frac{\partial(Uu')}{\partial(\frac{L}{U}t')} + (Uu') \frac{\partial(Uu')}{\partial(Lx')} + (Uv') \frac{\partial(Uu')}{\partial(Ly')} + (Uw') \frac{\partial(Uu')}{\partial(Lz')} = -\frac{1}{\rho} \frac{\partial(\rho U^2 p')}{\partial(Lx')} + \nu \left[\frac{\partial^2}{\partial(L^2 x'^2)} + \frac{\partial^2}{\partial(L^2 y'^2)} + \frac{\partial^2}{\partial(L^2 z'^2)} \right] (Uu')$$

$$\Rightarrow \frac{U}{L} \frac{\partial u'}{\partial t'} + \frac{U}{L} u' \frac{\partial u'}{\partial x'} + \frac{U}{L} v' \frac{\partial u'}{\partial y'} + \frac{U}{L} w' \frac{\partial u'}{\partial z'} = -\frac{U}{L} \frac{\partial p'}{\partial x'}$$

$$+ \frac{\nu U}{L^2} \left(\frac{\partial^2}{\partial x'^2} + \frac{\partial^2}{\partial y'^2} + \frac{\partial^2}{\partial z'^2} \right) u'$$

$$\Rightarrow \frac{U}{L} \left(\frac{\partial u'}{\partial t'} + u' \frac{\partial u'}{\partial x'} + v' \frac{\partial u'}{\partial y'} + w' \frac{\partial u'}{\partial z'} \right) = -\frac{U}{L} \frac{\partial p'}{\partial x'} + \frac{\nu U}{L^2} \left(\frac{\partial^2}{\partial x'^2} + \frac{\partial^2}{\partial y'^2} + \frac{\partial^2}{\partial z'^2} \right) u'$$

$$\left(\therefore \frac{\partial u}{\partial t} \approx \frac{U}{L} \frac{\partial u'}{\partial t'} \right)$$

Dividing by $\frac{U^2}{L}$, we have,

$$\frac{\partial u'}{\partial t'} + u' \frac{\partial u'}{\partial x'} + v' \frac{\partial u'}{\partial y'} + w' \frac{\partial u'}{\partial z'} = -\frac{\partial p'}{\partial x'} + \frac{\nu}{LU} \left(\frac{\partial^2 u'}{\partial x'^2} + \frac{\partial^2 u'}{\partial y'^2} + \frac{\partial^2 u'}{\partial z'^2} \right)$$

$$\Rightarrow \frac{\partial u'}{\partial t'} + u' \frac{\partial u'}{\partial x'} + v' \frac{\partial u'}{\partial y'} + w' \frac{\partial u'}{\partial z'} = -\frac{\partial p'}{\partial x'} + \frac{1}{R} \left(\frac{\partial^2 u'}{\partial x'^2} + \frac{\partial^2 u'}{\partial y'^2} + \frac{\partial^2 u'}{\partial z'^2} \right)$$

where $R = \frac{LU}{\nu}$ is the Reynolds number.

Similarly, the equations of motion along y and z axes are given by

$$\frac{\partial v'}{\partial t'} + u' \frac{\partial v'}{\partial x'} + v' \frac{\partial v'}{\partial y'} + w' \frac{\partial v'}{\partial z'} = -\frac{\partial p'}{\partial y'} + \frac{1}{R} \left(\frac{\partial^2 v'}{\partial x'^2} + \frac{\partial^2 v'}{\partial y'^2} + \frac{\partial^2 v'}{\partial z'^2} \right)$$

and $\frac{\partial w'}{\partial t'} + u' \frac{\partial w'}{\partial x'} + v' \frac{\partial w'}{\partial y'} + w' \frac{\partial w'}{\partial z'} = -\frac{\partial p'}{\partial z'} + \frac{1}{R} \left(\frac{\partial^2 w'}{\partial x'^2} + \frac{\partial^2 w'}{\partial y'^2} + \frac{\partial^2 w'}{\partial z'^2} \right)$

and the equation continuity

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0 \quad (\text{in non-dimensional form})$$

$$\frac{\partial u'}{\partial x'} + \frac{\partial v'}{\partial y'} + \frac{\partial w'}{\partial z'} = 0$$

Navier-Stokes equation is highly non-linear. It can be solved -

- (i) exactly if non-linear terms cancelled out
- (ii) when the non-linear term neglected retained partially with linear terms
- (iii) when the equations are approximated to convenient forms. (Boundary layer approximation).

Besides, there are few well-known problems which are exactly solved with non-linear terms.

- (i) Two dimensional steady flow between non-parallel

plates (Jeffery-Hamel problem)

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(ii) Motion due to rotating disk [Karman's problem]

(iii) Stagnatic flow [Hiemenz problem]

(iv) Flow from a circular hole [Lasdoll squire problem]

2.11:

The Reynolds number:

We consider the Navier-Stokes equation is

$$\frac{\partial \vec{q}}{\partial t} + \frac{1}{2} \nabla q^{\vee} - \vec{q} \times \vec{\zeta} = \vec{F} - \frac{1}{\rho} \nabla p + \nu \nabla^2 \vec{q} \quad \rightarrow (1)$$

If the external force \vec{F} is conservative, $\vec{F} = -\nabla \Omega$

and (1) can be written as

$$\begin{aligned} \frac{\partial \vec{q}}{\partial t} + \frac{1}{2} \nabla q^{\vee} - \vec{q} \times \vec{\zeta} &= -\nabla \Omega - \frac{1}{\rho} \nabla p + \nu \nabla^2 \vec{q} \\ &= -\nabla \left(\Omega + \frac{p}{\rho} \right) + \nu \nabla^2 \vec{q} \quad \rightarrow (11) \end{aligned}$$

If we take the external force to produce hydrostatic pressure p_0 then for fluid at rest, we find

$$-\nabla \left(\Omega + \frac{p_0}{\rho} \right) = 0 \quad \text{i.e. } p_0 = -\rho \Omega$$

If we consider the pressure p as the sum of hydrostatic pressure p_0 and hydrodynamic pressure p_1 i.e. $p = p_0 + p_1$, $p_0 = -\rho \Omega$, then equation (11) becomes,

$$\frac{\partial \vec{q}}{\partial t} + \frac{1}{2} \nabla q^{\vee} - \vec{q} \times \vec{\zeta} = -\frac{1}{\rho} \nabla p_1 + \nu \nabla^2 \vec{q}$$

Multiplying by ρ

$$\rho \left(\frac{\partial \vec{q}}{\partial t} + \frac{1}{2} \nabla q^{\vee} - \vec{q} \times \vec{\zeta} \right) = -\nabla p_1 + \mu \nabla^2 \vec{q} \quad ; \quad \because \mu = \rho \nu$$

This equation consists of three types of forces

(a) effective force $\rho \left\{ \frac{\partial \vec{q}}{\partial t} + \frac{1}{2} \nabla q^2 - \vec{q} \times \vec{\zeta} \right\}$

(b) the pressure force $-\nabla p$

(c) the viscous force $\mu \nabla^2 \vec{q}$

i.e. Effective force = pressure force + viscous force.

Now, the dimension of the terms of

(a) is $\rho \frac{U^2}{L}$, $U = \text{typical velocity}$

$L = \text{length}$

$\rho = \text{density}$.

and the dimension of

$\mu \nabla^2 \vec{q} = \mu \frac{U}{L^2}$

$\therefore \rho \frac{\partial \vec{q}}{\partial t} = \frac{\rho U^2}{L}$

$\rho \frac{1}{2} \nabla q^2 = \dots$

$\rho \vec{q} \times \vec{\zeta} = \dots$

Therefore the ratio of effective force and viscous force is

$\frac{\rho \frac{U^2}{L}}{\mu \frac{U}{L^2}} = \frac{UL}{\nu}$, which represents a number and is called Reynold number.

Two forces are similar when R is equal for both this principle was first enumerated by Reynold in connection with his investigation into the flow through pipes and is known as Reynold's principle of similarity.