

If the bed be inclined with the angle θ to the horizon, then equation for u along the bed is

$$\mu \frac{\partial^2 u}{\partial y^2} + \left(\rho g \sin \theta - \frac{\partial p}{\partial x} \right) = 0$$

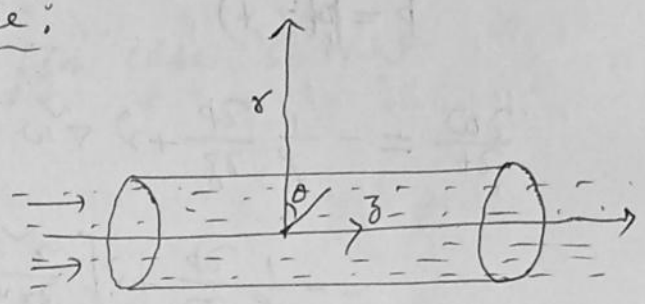
In this case,

$$\rho g \sin \theta - \frac{\partial p}{\partial x} = -\rho k \text{ (constant)}$$

and u is same (can be determined) as before. #

2.13 Flow through a circular pipe:

Let us consider viscous liquid through a circular pipe whose axis is in the direction of z -axis



and r -axis is taken perpendicular to it as shown in figure.

Let u, v, w be the velocity components along x, y and z axes respectively, then

$$u = v = 0, w \neq 0$$

Now, consider the equation of continuity

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0$$

$$\Rightarrow \frac{\partial w}{\partial z} = 0 \quad \text{--- (1)}$$

and equation of motion

$$\rho \left[\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} + w \frac{\partial u}{\partial z} \right] = -\frac{\partial p}{\partial x} + \mu \left[\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} + \frac{\partial^2 u}{\partial z^2} \right]$$

$$\rho \left[\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} + w \frac{\partial v}{\partial z} \right] = -\frac{\partial p}{\partial y} + \mu \left[\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} + \frac{\partial^2 v}{\partial z^2} \right]$$

$$\frac{\partial \omega}{\partial t} + u \frac{\partial \omega}{\partial r} + \frac{v}{r} \frac{\partial \omega}{\partial \theta} + w \frac{\partial \omega}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \left[\nabla^2 \omega \right],$$

$$\text{where } \nabla^2 \equiv \frac{\partial^2}{\partial r^2} + \frac{1}{r} \frac{\partial}{\partial r} + \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} + \frac{\partial^2}{\partial z^2}$$

$$\text{e. } -\frac{1}{\rho} \frac{\partial p}{\partial r} = 0 \quad \rightarrow (i)$$

$$-\frac{1}{\rho} \frac{\partial p}{r \partial \theta} = 0 \quad \rightarrow (ii)$$

$$\frac{\partial \omega}{\partial t} = -\frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \nabla^2 \omega \quad \rightarrow (iv)$$

From the above equation, we have,

$$\omega = \omega(r, t)$$

$$p = p(z, t)$$

$$\frac{\partial \omega}{\partial t} = -\frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \nabla^2 \omega$$

$$= -\frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \left[\frac{\partial^2}{\partial r^2} + \frac{1}{r} \frac{\partial}{\partial r} + \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} + \frac{\partial^2}{\partial z^2} \right] \omega$$

$$= -\frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \left[\frac{\partial^2 \omega}{\partial r^2} + \frac{1}{r} \frac{\partial \omega}{\partial r} \right] \quad \rightarrow (v)$$

This equation contains two unknowns ω, p .

Now, we multiply both sides of $2\pi r dr$ and integrating from $r=0$ to $r=a$, where a is the radius of the pipe.

$$\int_0^a \frac{\partial \omega}{\partial t} \cdot 2\pi r dr = -\int_0^a -\frac{1}{\rho} \frac{\partial p}{\partial z} \cdot 2\pi r dr + \int_0^a \nu \left[\frac{\partial^2 \omega}{\partial r^2} + \frac{1}{r} \frac{\partial \omega}{\partial r} \right] \cdot 2\pi r dr.$$

$$\Rightarrow \frac{\partial}{\partial t} \int_0^a \omega \cdot 2\pi r dr = -\int_0^a \frac{\partial p}{\partial z} \cdot 2\pi r dr + \int_0^a \frac{\mu}{\rho} \frac{1}{r} \left[\frac{\partial}{\partial r} \left\{ r \frac{\partial \omega}{\partial r} \right\} \right] \cdot 2\pi r dr.$$

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$$\Rightarrow \frac{\partial m}{\partial t} = - \frac{\partial p}{\partial z} \int_0^a 2\pi r dr + \mu \cdot 2\pi \left[r \frac{\partial \omega}{\partial r} \right]_0^a, \quad \text{where } m = \int_0^a \omega \cdot 2\pi r dr$$

$$= - \frac{\partial p}{\partial z} \left[\pi r^2 \right]_0^a + 2\pi \mu a \left(\frac{\partial \omega}{\partial r} \right)_{r=a}$$

$$\Rightarrow \frac{\partial m}{\partial t} = - \pi a^2 \frac{\partial p}{\partial z} + 2\pi a \mu \left[\frac{\partial \omega}{\partial r} \right]_{r=a}$$

\Rightarrow Rate of change of flow of total mass = - pressure gradient on the

cross-section + viscous stress on the boundary.

Case (I):

Let us consider the steady flow case. So that

$$p = p(z), \quad \omega = \omega(r)$$

then the equation (v) reduces to

$$- \frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial \omega}{\partial r} \right) \right] = 0 \quad \rightarrow (vi)$$

Now, the 1st part of equation (vi) depends on z only and the 2nd part depends on r only. Therefore the equation is compatible for solution if

$$\frac{1}{\rho} \frac{\partial p}{\partial z} = A.$$

From (vi), we have,

$$-A + \nu \frac{\partial}{\partial r} \left(r \frac{\partial \omega}{\partial r} \right) = 0$$

$$\Rightarrow \frac{\partial}{\partial r} \left(r \frac{\partial \omega}{\partial r} \right) = \frac{Ar}{\nu}$$

$$\Rightarrow \frac{d}{dr} \left(r \frac{d\omega}{dr} \right) = \frac{Ar}{\nu}$$

$$- \frac{dp}{dz} = \rho A$$

The -ve sign is taken as convention.

$$\frac{d}{dr} \left(r \frac{d\omega}{dr} \right) = - \frac{\rho r}{\mu}$$

Here $\vec{\omega} = \vec{z}$.

$$r \frac{dw}{dr} = \frac{Ar^{\nu}}{2\nu} + B \text{ (say)}$$

$$\Rightarrow \frac{dw}{dr} = \frac{A}{2\nu} \frac{r^{\nu}}{r} + \frac{B}{r}$$

$$\Rightarrow W = \frac{A}{2\nu} \frac{r^{\nu}}{2} + B \log r + c$$

$$\Rightarrow W = \frac{A}{4\nu} \cdot r^{\nu} + B \log r + c, \text{ where } A, B, c \text{ are constants}$$

Boundary conditions are

(a) $w = 0$ at $r = a$

and (b) w is continuous when $0 < r \leq a$.

From condition (b) we have $B = 0$, otherwise discontinuity will come along the axis of the pipe.

Thus from condition (a)

$$0 = \frac{A}{2\nu} \cdot \frac{a^{\nu}}{2} + c$$

$$\Rightarrow c = -\frac{Aa^{\nu}}{4\nu}$$

Hence, $w = \frac{A}{4\nu} (r^{\nu} - a^{\nu})$

Now, we are to determine A . If pressure gradient $\frac{\partial p}{\partial z}$ is known then A is known. Otherwise A can be found if the total flow of mass across any section is known.

Now, total flow of mass across a section of the pipe is

$$m = \int_0^a 2\pi r w \rho dr = \int_0^a \frac{2\pi \rho A}{4\nu} (r^{\nu} - a^{\nu}) r dr$$

$$= \frac{2\pi \rho A}{4\nu} \left[\frac{r^4}{4} - a^{\nu} \frac{r^{\nu}}{2} \right]_0^a$$

$$= \frac{2\pi\rho A}{4\nu} \left[\frac{a^4}{4} - \frac{a^4}{2} \right]$$

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$$= \frac{\pi\rho A}{2\nu} \left(-\frac{a^4}{4} \right)$$

$$= -\frac{A\pi\rho a^4}{8\nu}$$

Since m is positive, therefore A is -ve (i.e. pressure gradient is -ve) and hence

$$A = -\frac{8\nu m}{\pi\rho a^4}$$

$$\therefore W = -\frac{8\nu m}{\pi\rho a^4} \cdot \frac{1}{4\nu} (r^2 - a^2)$$

$$= \frac{2m}{\pi\rho a^4} (a^2 - r^2)$$

This is a parabolic velocity profile.

At $r=a$, $w=0$

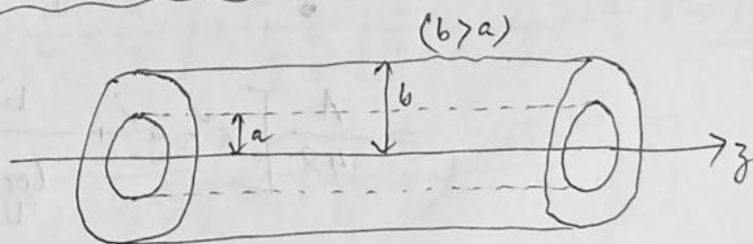
and at $r=0$, $w = \frac{2m}{\pi\rho a^2}$

i.e. w is maximum.

[This experiment was done by Hagen in 1839 and afterwards by Poiseuille in 1840 and the flow is known as Hagen-Poiseuille flow.]

Case (II): Steady flow between coaxial circular pipes:

We get the solution of w as



$$W = \frac{A}{4\nu} r^2 + B \log r + C$$

Let the flow takes places between two coaxial circular pipes of radii a and b , $b > a$ as shown in figure. The boundary conditions on w are

$$w = 0 \text{ when } r = a$$

$$w = 0 \text{ when } r = b.$$

when $r = a$, we have,

$$0 = \frac{A}{4\mu} a^2 + B \log a + C$$

When $r = b$, we have,

$$0 = \frac{A}{4\mu} b^2 + B \log b + C$$

Subtracting we get,

$$0 = \frac{A}{4\mu} (a^2 - b^2) + B [\log a - \log b]$$

$$\Rightarrow B \log \frac{a}{b} = \frac{A}{4\mu} (b^2 - a^2)$$

$$\Rightarrow B = \frac{\frac{A}{4\mu} (b^2 - a^2)}{\log \frac{a}{b}}$$

and $C = -\frac{A a^2}{4\mu} - \frac{A}{4\mu} \frac{b^2 - a^2}{\log \frac{a}{b}} \cdot \log a$

Therefore

$$w = \frac{A}{4\mu} r^2 + \frac{A}{4\mu} \frac{(b^2 - a^2)}{\log \frac{a}{b}} \log r - \frac{A a^2}{4\mu} - \frac{A}{4\mu} \frac{b^2 - a^2}{\log \frac{a}{b}} \cdot \log a$$

$$= \frac{A}{4\mu} \left[r^2 - a^2 + \frac{b^2 - a^2}{\log \frac{a}{b}} \log \frac{r}{a} \right]$$

When the pressure gradient is known, then A is known,

otherwise A is to be determined from the value of total flow of mass across any section of the pipe.

$$\begin{aligned} \therefore m &= \int_{r=a}^b w 2\pi \rho r dr \\ &= 2\pi \rho \int_{r=a}^b \frac{A}{4r} \left[r^{\sqrt{}} - a^{\sqrt{}} + \frac{b^{\sqrt{}} - a^{\sqrt{}}}{\log \frac{a}{b}} \log \frac{r}{a} \right] r dr \\ &= 2\pi \rho \frac{A}{4r} \left\{ \left[\frac{r^4}{4} - a^{\sqrt{}} \frac{r^{\sqrt{}}}{2} \right]_{r=a}^b + \frac{b^{\sqrt{}} - a^{\sqrt{}}}{\log \frac{a}{b}} \int_{r=a}^b \log \frac{r}{a} r dr \right\} \end{aligned}$$

Now, $\int_{r=a}^b [\log r - \log a] r dr$

$$= \int_{r=a}^b r \log r dr - \int_{r=a}^b \log a \cdot r dr$$

$$\begin{aligned} &= \left[\log r \cdot \frac{r^{\sqrt{}}}{2} \right]_a^b - \int_a^b \frac{1}{r} \cdot \frac{r^{\sqrt{}}}{2} dr - \log a \left[\frac{r^{\sqrt{}}}{2} \right]_a^b \\ &= \frac{b^{\sqrt{}}}{2} \log b - \frac{a^{\sqrt{}}}{2} \log a - \frac{1}{2} \left[\frac{r^{\sqrt{}}}{2} \right]_a^b - \log a \left[\frac{b^{\sqrt{}} - a^{\sqrt{}}}{2} \right] \\ &= \frac{b^{\sqrt{}}}{2} \log b - \frac{a^{\sqrt{}}}{2} \log a - \frac{1}{4} (b^{\sqrt{}} - a^{\sqrt{}}) - \frac{b^{\sqrt{}}}{2} \log a + \frac{a^{\sqrt{}}}{2} \log a \\ &= -\frac{1}{4} (b^{\sqrt{}} - a^{\sqrt{}}) + \frac{b^{\sqrt{}}}{2} (\log b - \log a) \end{aligned}$$

$$\begin{aligned} \therefore m &= 2\pi \rho \frac{A}{4r} \left\{ \frac{b^4}{4} - \frac{a^{\sqrt{}} b^{\sqrt{}}}{2} - \frac{a^4}{4} + \frac{a^4}{2} \right\} + \frac{b^{\sqrt{}} - a^{\sqrt{}}}{\log \frac{a}{b}} \left[\frac{b^{\sqrt{}}}{2} (\log b - \log a) - \frac{1}{4} (b^{\sqrt{}} - a^{\sqrt{}}) \right] \\ &= \frac{A \pi \rho}{2r} \left[\frac{b^4 - a^4}{4} - \frac{a^{\sqrt{}}}{2} (b^{\sqrt{}} - a^{\sqrt{}}) + \frac{b^{\sqrt{}} - a^{\sqrt{}}}{\log \frac{a}{b}} \left\{ \frac{b^{\sqrt{}}}{2} (\log b - \log a) - \frac{b^{\sqrt{}} - a^{\sqrt{}}}{4} \right\} \right] \end{aligned}$$

Thus A may be obtained in terms of m and substituting A we may obtain w also.