

Ex. Establish the formula $\frac{1}{8} \frac{\pi a^4}{\mu L} (P_1 - P_2)$

for the rate of steady flow of an incompressible liquid through a uniform circular pipe of radius 'a', P_1 and P_2 being the pressure of two section of the pipe distant 'l' apart.

Solⁿ. We consider the steady flow of an incompressible viscous fluid through a circular pipe of radius 'a'. Let u, v, w be the velocity components in the direction of r, θ, z respectively. Then the boundary conditions are

$$\left. \begin{aligned} u=0, v=0, w=0 \text{ at } r=a \\ u=0, v=0, \frac{dw}{dr}=0 \text{ at } r=0 \end{aligned} \right\} \rightarrow (i)$$

the axis of a pipe is taken as z-axis. In view of these conditions we take

$$u=0, v=0, w=w(r).$$

Now, the equation of continuity and motion are

$$\frac{\partial u}{\partial r} + \frac{u}{r} + \frac{\partial v}{\partial \theta} + \frac{\partial w}{\partial z} = 0$$

$$u \frac{\partial u}{\partial r} + \frac{v}{r} \frac{\partial u}{\partial \theta} + w \frac{\partial u}{\partial z} - \frac{v^2}{r} = -\frac{1}{\rho} \frac{\partial p}{\partial r} + \gamma \left[\nabla^2 u - \frac{2}{r^2} \frac{\partial v}{\partial \theta} - \frac{u}{r^2} \right]$$

$$u \frac{\partial v}{\partial r} + \frac{v}{r} \frac{\partial v}{\partial \theta} + w \frac{\partial v}{\partial z} + \frac{uv}{r} = -\frac{1}{\rho} \frac{\partial p}{\partial \theta} + \gamma \left[\nabla^2 v + \frac{2}{r^2} \frac{\partial u}{\partial \theta} - \frac{v}{r^2} \right]$$

$$u \frac{\partial w}{\partial r} + \frac{v}{r} \frac{\partial w}{\partial \theta} + w \frac{\partial w}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial z} + \gamma \nabla^2 w,$$

$$\text{where } \nabla^2 \equiv \frac{\partial^2}{\partial r^2} + \frac{1}{r} \frac{\partial}{\partial r} + \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} + \frac{\partial^2}{\partial z^2}$$

In this case, the equation reduces to

$$\frac{\partial w}{\partial z} = 0$$

$$-\frac{1}{\rho} \frac{\partial p}{\partial r} = 0$$

$$-\frac{1}{\rho} \frac{\partial p}{r \partial \theta} = 0$$

$$-\frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \left[\frac{\partial^2 \omega}{\partial r^2} + \frac{1}{r} \frac{\partial \omega}{\partial r} \right] = 0$$

i.e. the equation of motion in the direction of r and θ gives that the hydrostatic pressure P is independent of r and θ and the equation of motion in the z -direction is given by

$$\mu \left[\frac{d^2 \omega}{dr^2} + \frac{1}{r} \frac{d\omega}{dr} \right] = \frac{dp}{dz} \longrightarrow (ii)$$

The left side of this equation is a function of r only and the right side is a function of z -only, hence both must be a constant.

$$\text{From (ii)} \quad \frac{d}{dr} \left[r \frac{d\omega}{dr} \right] = \frac{r}{\mu} \frac{dp}{dz}$$

$$\text{Integrating,} \quad r \frac{d\omega}{dr} = \frac{1}{\mu} \frac{dp}{dz} \cdot \frac{r^2}{2} + B \longrightarrow (iii)$$

$$\Rightarrow \frac{d\omega}{dr} = \frac{1}{2\mu} \frac{dp}{dz} r + \frac{B}{r}$$

Again, integrating,

$$\omega = \frac{1}{2\mu} \frac{dp}{dz} \cdot \frac{r^3}{2} + B \log r + C \longrightarrow (iv)$$

At $r=0$

$$(iii) \Rightarrow 0 = 0 + B \Rightarrow B = 0$$

At $r=0$

$$(iv) \Rightarrow 0 = \frac{1}{2\mu} \frac{dp}{dz} \cdot \frac{a^3}{2} + B \log a + C$$

$$\Rightarrow C = -\frac{a^3}{4\mu} \frac{dp}{dz}$$

$$\omega = \frac{1}{2\mu} \frac{dP}{dz} \frac{r^2}{2} - \frac{a^2}{4\mu} \frac{dP}{dz}$$

$$= \frac{1}{4\mu} \frac{dP}{dz} (r^2 - a^2)$$

The maximum velocity is on the axis and is given by

$$\omega = \frac{a^2}{4\mu} \left(-\frac{dP}{dz}\right)$$

The mean velocity is given by

$$\bar{\omega} = \frac{\omega}{2}$$

$$\bar{\omega} = \frac{a^2}{8\mu} \left(-\frac{dP}{dz}\right)$$

And the volume of fluid flowing through the pipe per unit time is given by

$$Q = \pi a^2 \bar{\omega} = \frac{\pi a^4}{8\mu} \left(-\frac{dP}{dz}\right)$$

If the length of the pipe is 'l' and the pressure at the ends be P₁ and P₂ then,

$$-\frac{dP}{dz} = \frac{P_1 - P_2}{l}$$

and then the flux through the pipe is

$$Q = \frac{\pi a^4}{8l\mu} (P_1 - P_2) \quad \#$$

[This flow is known as Hagan - Poiseuille flow]

Ex. Show that in the two-dimensional motion of a viscous liquid, acted on by a conservative system of forces the stream function satisfies the equation

$$\left(\nu \nabla^2 - \frac{\partial}{\partial t}\right) \nabla^2 \psi = \frac{\partial(\psi, \nabla^2 \psi)}{\partial(x, y)}$$

where ν is the kinematic viscosity.

Solⁿ.

The Navier-Stokes equation for incompressible viscous fluid (under the action of conservative system of forces) is given by

$$\frac{\partial \vec{c}}{\partial t} + (\vec{c} \cdot \nabla) \vec{c} - (\vec{c} \cdot \nabla) \vec{c} = \nu \nabla^2 \vec{c} \quad \text{--- (1)}$$

$$\text{where } \vec{c} = \text{curl } \vec{q}$$

Since the motion is two-dimensional motion of a viscous fluid, therefore

$$\vec{q} = (u, v, 0), \quad \vec{c} = (0, 0, f)$$

$$\vec{c} = \text{curl } \vec{q} = \nabla \times \vec{q} = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ \frac{\partial}{\partial x} & \frac{\partial}{\partial y} & \frac{\partial}{\partial z} \\ u & v & 0 \end{vmatrix}$$

Now from (i) we have,

$$\begin{aligned} \left(u \frac{\partial}{\partial x} + v \frac{\partial}{\partial y}\right) f &= \nu \nabla^2 f - \frac{\partial f}{\partial t} \\ &= \left(\nu \nabla^2 - \frac{\partial}{\partial t}\right) f \quad \text{--- (ii)} \end{aligned}$$

Now, if the stream function ψ exists then

$$u = -\frac{\partial \psi}{\partial y}, \quad v = \frac{\partial \psi}{\partial x}$$

$$f = \frac{\partial v}{\partial x} - \frac{\partial u}{\partial y} = \frac{\partial^2 \psi}{\partial x^2} + \frac{\partial^2 \psi}{\partial y^2} = \nabla^2 \psi$$

∴ (ii) can be written as

$$\begin{aligned}
(\nabla^2 \tilde{\psi} - \frac{\partial \tilde{\psi}}{\partial t}) &= u \left(\frac{\partial f}{\partial x} \right) + v \left(\frac{\partial f}{\partial y} \right) \\
&= \frac{\partial \psi}{\partial x} \frac{\partial}{\partial y} (\nabla^2 \tilde{\psi}) - \frac{\partial \psi}{\partial y} \frac{\partial}{\partial x} (\nabla^2 \tilde{\psi}) \\
&= \frac{\partial \psi}{\partial x} (\psi) \frac{\partial}{\partial y} (\nabla^2 \tilde{\psi}) - \frac{\partial \psi}{\partial y} (\psi) \frac{\partial}{\partial x} (\nabla^2 \tilde{\psi}) \\
&= \frac{\partial(\psi, \nabla^2 \tilde{\psi})}{\partial(x, y)}
\end{aligned}$$

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Ex. A liquid occupying the space between two coaxial circular cylinder is acted upon by a force $\frac{c}{r}$ per unit mass, where r is the distance from the axis, the lines of force being circles round the axis. Prove that in the steady motion the velocity at any point is given by the formula

$$\frac{1}{2} \frac{c}{\nu} \left[\frac{b^2}{r} \cdot \frac{(r^2 - a^2)}{(b^2 - a^2)} \log \frac{b}{a} - r \cdot \log \frac{r}{a} \right]$$

where ν is the coefficient of kinematic viscosity and a, b are the two radii.

Solⁿ. Let z -axis be the axis of the cylinders. If u, v, w be the velocity components along r, θ, z directions.

Then $u = w = 0, v = r\Omega \rightarrow (i)$

where Ω is the angular velocity of the liquid at any point.