

2012

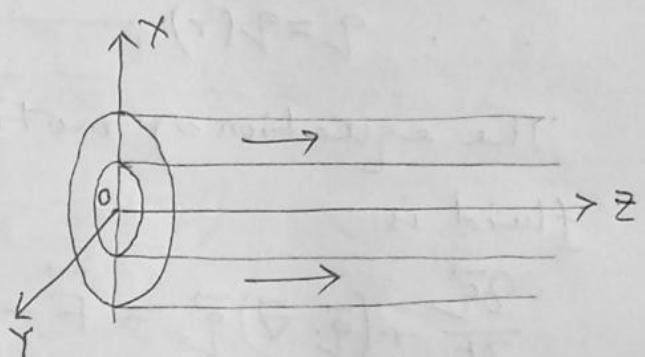
(79)

Ex. A viscous liquid flows steadily parallel to the axis in the annular space between two coaxial cylinders of radii a and na ($n > 1$), show that the rate of discharge is

$$\frac{\pi P a^4}{8\mu} \left\{ n^4 - 1 - \frac{(n^2 - 1)^2}{\log n} \right\}$$

where P is the pressure gradient and μ is the coefficient of viscosity.

Sol) Take z -axis along the common axis of the cylinders and x -axis and y -axis as shown in the figure.



Let $\vec{q} = (q_x, q_y, q_z)$ be the fluid velocity at the point $P(r, \theta, z)$ in the fluid.

since the flow is parallel to z -axis

$$\begin{aligned} \therefore q_x &= 0, \\ q_y &= 0 \end{aligned} \quad \left. \right\}$$

and let $q_z = v$

\therefore The equation of continuity is

$$\nabla \cdot \vec{q} = 0$$

$$\Rightarrow \frac{1}{r \cdot r \cdot 1} \left[\frac{\partial}{\partial r} (r \cdot 1 \cdot q_r) + \frac{\partial}{\partial \theta} (1 \cdot 1 \cdot q_\theta) + \frac{\partial}{\partial z} (1 \cdot r \cdot q_z) \right] = 0$$

$$\Rightarrow \frac{1}{r} \frac{\partial}{\partial r} (r q_z) = 0$$

$$\Rightarrow \frac{\partial q_z}{\partial r} = 0 \quad \Rightarrow \frac{\partial v}{\partial r} = 0$$

$\therefore \vec{q}$ is independent of θ .

Due to actual symmetric,

$\therefore \vec{q}$ is independent of r .

Since the motion is steady

$\therefore \vec{q}$ is independent of 't'.

$$\therefore \vec{q} = \vec{q}(r) \rightarrow ①$$

The equation of motion for an incompressible viscous fluid is

$$\frac{\partial \vec{q}}{\partial t} + (\vec{q} \cdot \vec{\nabla}) \vec{q} = \vec{F} - \frac{1}{\rho} \vec{\nabla} p - 2 \vec{\nabla} \times \vec{\xi}$$

$$\Rightarrow \frac{\partial \vec{q}}{\partial t} + \frac{1}{2} \vec{\nabla} q^2 - \vec{q} \times \vec{\xi} = \vec{F} - \frac{1}{\rho} \vec{\nabla} p - 2 \vec{\nabla} \times \vec{\xi} \rightarrow ②$$

Since the motion is steady and there is no body force acting on the fluid.

$$\left. \begin{array}{l} \frac{\partial \vec{q}}{\partial t} = 0 \\ \vec{F} = 0 \end{array} \right\} \rightarrow ③$$

By virtue of ③ the equation ② reduces to

$$\frac{1}{2} \vec{\nabla} q^2 - \vec{q} \times \vec{\xi} = - \frac{1}{\rho} \vec{\nabla} p - 2 \vec{\nabla} \times \vec{\xi} \rightarrow ④$$

$$\vec{\nabla} q^2 = \hat{r} \frac{\partial q^2}{\partial r} = \hat{r} \frac{d q^2}{dr} = \hat{r} \cdot 2 q^2 \hat{r} = 2 q^2 \hat{r} \rightarrow ⑤$$

$$\vec{\xi} = \vec{\nabla} \times \vec{q} = \frac{1}{1.r.1} \begin{vmatrix} \hat{r} & \hat{r} & \hat{z} \\ \frac{\partial}{\partial r} & \frac{\partial}{\partial \theta} & \frac{\partial}{\partial z} \\ 0 & 0 & q \end{vmatrix} = - \hat{\theta} q'$$

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$$\vec{\nabla} \times \vec{q} = \frac{1}{r \cdot r \cdot 1} \begin{vmatrix} \hat{r} & r\hat{\theta} & \hat{z} \\ \frac{\partial}{\partial r} & \frac{\partial}{\partial \theta} & \frac{\partial}{\partial z} \\ 0 & -rz' & 0 \end{vmatrix}$$

$$= -\frac{\hat{z}}{r} \frac{d}{dr} (rz') \longrightarrow (5)$$

$$\vec{q} \times \vec{q}' = q \hat{z} \times (-q' \hat{z})$$

$$= q q' (\hat{z} \times \hat{z}) = q q' \hat{z} \longrightarrow (6)$$

using (4), (5) & (6) in (A), we get,

$$qq' \hat{z} - qq' \hat{z} = -\frac{1}{\rho} \vec{\nabla} p + \mu \frac{\hat{z}}{r} \frac{d}{dr} (rz')$$

$$\Rightarrow \vec{\nabla} p = \mu \frac{\hat{z}}{r} \frac{d}{dr} (rz')$$

$$\Rightarrow \frac{\partial p}{\partial r} + \frac{\hat{\theta}}{r} \frac{\partial p}{\partial \theta} + \frac{\hat{z}}{r} \frac{\partial p}{\partial z} = -\frac{\mu}{r} \frac{d}{dr} (rz') \hat{z} \longrightarrow (7)$$

By equating the coefficient of \hat{r} , $\hat{\theta}$ and \hat{z} in the equation (7) we derive,

$$\frac{\partial p}{\partial r} = 0 \longrightarrow (8.1)$$

$$\frac{\partial p}{\partial \theta} = 0 \longrightarrow (8.2)$$

$$\frac{\partial p}{\partial z} = -\frac{\mu}{r} \frac{d}{dr} (rz') \longrightarrow (8.3)$$

(8.1) and (8.2) shows that p is independent of r and θ .

further as the motion is steady

p is independent of t

$$\therefore p = p(z) \longrightarrow (9)$$

By the use of the equation (9), the equation (8.3) reduces to

$$\frac{dp}{dq} = \frac{\mu}{r} \frac{d}{dr}(rq')$$

\Rightarrow a function of q = a function of r

$$\frac{\mu}{r} \frac{d}{dr}(rq') = \frac{dp}{dq} = \text{a constant} = -P(\text{say})$$

$$\Rightarrow \frac{d}{dr}(rq') = -\frac{Pr}{\mu}$$

$$\Rightarrow rq' = -\frac{Pr}{2\mu} + A$$

$$\Rightarrow \frac{dq'}{dr} = -\frac{Pr}{2\mu} + \frac{A}{r}$$

$$\Rightarrow q' = -\frac{Pr^2}{4\mu} + A \log r + B \rightarrow (10)$$

where A and B are two arbitrary constants to be determined from the flow problem under considerations. The boundary condition of the flow problem are

$$q' = 0 \text{ at } r = a \rightarrow (11.1)$$

$$q' = 0 \text{ at } r = na \rightarrow (11.2)$$

Subjecting (11.1) and (11.2) in (10), we get,

$$0 = -\frac{Pa^2}{4\mu} + A \log a + B \rightarrow (12.1)$$

$$0 = -\frac{Pn^2 a^2}{4\mu} + A \log na + B \rightarrow (12.2)$$

$$\therefore (12.2) - (12.1) \Rightarrow \frac{Pa^2}{4\mu} - \frac{Pn^2 a^2}{4\mu} + A \log n = 0$$

$$\Rightarrow A \log n = \frac{Pa^2}{4\mu} (n^2 - 1)$$

$$\Rightarrow A = \frac{Pa^2(n^2 - 1)}{4\mu \log n}$$

$$(12.1) \Rightarrow B = -\frac{\rho a^{\sqrt{v}}}{4\mu} - A \log a$$

(83)

$$= -\frac{\rho a^{\sqrt{v}}}{4\mu} - \frac{\rho a^{\sqrt{v}}(n^{\sqrt{v}-1})}{4\mu \log n} \log a$$

$$\Rightarrow B = -\frac{\rho a^{\sqrt{v}}}{4\mu} \left[1 - (n^{\sqrt{v}-1}) \frac{\log a}{\log n} \right]$$

Let Q be the rate of discharge (rate of flow or flux)

$$\therefore Q = \int_{\theta=0}^{2\pi} \int_{r=a}^{na} q r \omega dr$$

$$= \int_{\theta=0}^{2\pi} d\theta \int_{r=a}^{na} \left[-\frac{\rho r^{\sqrt{v}}}{4\mu} + A \log r + B \right] r dr$$

$$= 2\pi \left[-\frac{\rho}{4\mu} \int_a^{na} r^3 dr + A \int_a^{na} r \log r dr + B \int_a^{na} r dr \right]$$

$$= 2\pi \left[A I_1 + B I_2 - \frac{\rho}{4\mu} I_3 \right] \rightarrow (13)$$

$$\text{Now, } I_1 = \int_a^{na} \log r \cdot r dr$$

$$= \log r \cdot \left[\frac{r^2}{2} \right]_a^{na} - \int_a^{na} \frac{1}{r} \cdot \frac{r^2}{2} dr$$

$$= \frac{1}{2} \left[n^{\sqrt{v}} \log na - a^{\sqrt{v}} \log a \right] - \frac{1}{2} \int_a^{na} r dr$$

$$= \frac{1}{2} \left[n^{\sqrt{v}} a^{\sqrt{v}} \log na - a^{\sqrt{v}} \log a \right] - \frac{1}{4} (n^{\sqrt{v}-1}) a^{\sqrt{v}}$$

$$= \frac{a^{\sqrt{v}}}{2} (n^{\sqrt{v}} \log na - \log a) - \frac{1}{4} (n^{\sqrt{v}-1}) a^{\sqrt{v}}$$

$$I_2 = \int_a^{na} r dr = \frac{1}{2} (n^2 a^2 - a^2) = \frac{1}{2} (n^2 - 1) a^2 \quad (84)$$

$$I_3 = \int_a^{na} r^3 dr = \frac{1}{4} (n^4 - 1) a^4.$$

$$\therefore Q = 2\pi [T_1 + T_2 - T_3]$$

$$T_1 = A I_1 = \frac{P a^2 (n^2 - 1)}{4\mu \log n} \left[\frac{a^2}{2} (n^2 \log n - \log a) - \frac{1}{4} (n^2 - 1) a^2 \right]$$

$$= \frac{P a^4 (n^2 - 1)}{16\mu \log n} \left[2(n^2 \log n - \log a) - (n^2 - 1) \right]$$

$$T_2 = B I_2 = \frac{P a^2}{4\mu} \left[1 - \frac{n^2 - 1}{\log n} \log a \right] \frac{1}{2} (n^2 - 1) a^2$$

$$= \frac{P a^4}{8\mu} \left[n^2 - 1 - \frac{(n^2 - 1)^2}{\log n} \log a \right]$$

$$T_3 = \frac{P}{4\mu} I_3 = \frac{P}{4\mu} \cdot \frac{1}{4} (n^4 - 1) a^4$$

$$= \frac{P a^4}{16\mu} (n^4 - 1)$$

$$\therefore Q = 2\pi \cdot \frac{P a^4}{16\mu} \left[\frac{n^2 - 1}{\log n} \left\{ 2(n^2 \log n - \log a) - (n^2 - 1) \right\} \right. \\ \left. + 2 \left\{ (n^2 - 1) - \frac{(n^2 - 1)^2 \log a}{\log n} \right\} - (n^4 - 1) \right]$$

$$= 2\pi \cdot \frac{P a^4}{16\mu} \left[\frac{(n^2 - 1)}{\log n} \left\{ 2(n^2 \log n - \log a) - (n^2 - 1) \right\} + 2(n^2 - 1) \right. \\ \left. - \frac{2(n^2 - 1)^2 \log a}{\log n} \right\} - (n^4 - 1) \right]$$

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$$= \frac{\pi Pa^4}{8\mu} \left[\frac{2(n^{\gamma}-1)(n^{\gamma}\log n - \log a) - (n^{\gamma}-1)^{\gamma} - 2(n^{\gamma}-1)^{\gamma}\log a + 2(n^{\gamma}-1) - (n^{\gamma}-1)}{\log n} \right]$$

$$= \frac{\pi Pa^4}{8\mu} \left[\frac{(n^{\gamma}-1) \{ 2n^{\gamma}\log n - 2\log a + n^{\gamma} + 1 - 2(n^{\gamma}-1)\log a \} + 2(n^{\gamma}-1) - (n^{\gamma}-1)}{\log n} \right]$$

$$= \frac{\pi Pa^4}{8\mu} \left[\frac{(n^{\gamma}-1) \{ 2n^{\gamma}\log n - n^{\gamma} + 1 \}}{\log n} + 2(n^{\gamma}-1) - (n^{\gamma}-1) \right]$$

$$= \frac{\pi Pa^4}{8\mu} \left[(n^{\gamma}-1) \left(2n^{\gamma} - \frac{n^{\gamma}-1}{\log n} \right) + 2(n^{\gamma}-1) - (n^{\gamma}-1) \right]$$

$$= \frac{\pi Pa^4}{8\mu} \left[2n^{\gamma} - t \frac{(n^{\gamma}-1)^{\gamma}}{\log n} + 2n^{\gamma} + 2n^{\gamma} - 2 - n^{\gamma} + 1 \right]$$

$$= \frac{\pi Pa^4}{8\mu} \left[n^{\gamma} - 1 - \frac{(n^{\gamma}-1)^{\gamma}}{\log n} \right]$$

#

(86)

Ex In a steady motion of a viscous liquid through a cylindrical pipe, show that the equation of motion becomes

$$\frac{\partial \tilde{w}}{\partial z^2} + \frac{\partial \tilde{w}}{\partial y^2} = -\frac{P}{\mu},$$

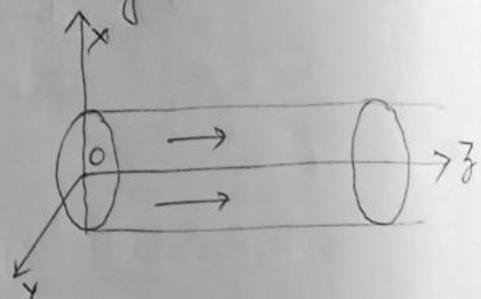
where μ is the viscosity, w , $-P$ are respectively the velocity and pressure gradient along the axis of the cylinder. Further show that in the case of an elliptic cylinder across section given by $\frac{x^2}{a^2} + \frac{y^2}{b^2} = 1$, the flux of the liquid through the cylinder is

$$Q = \frac{\pi}{4\mu} \frac{P a^3 b^3}{a^2 + b^2}$$

Also obtain the maximum velocity.

Sol:

Take z -axis along the axis of the pipe and other axes as shown in the figure.



Let $\vec{v} = (u, v, w)$ be the fluid velocity at the point $P(x, y, z)$ in the fluid,

In the present case since the fluid motion is parallel to z -axis

$$\therefore u = 0, v = 0$$

$$\therefore \vec{v} = w \hat{k}$$

The equation of continuity for an incompressible

fluid is $\vec{\nabla} \cdot \vec{q} = 0$

$$\Rightarrow \frac{\partial \omega}{\partial z} = 0$$

$\therefore \omega$ is independent of z .

Again, since the motion is steady

$\therefore \omega$ is independent of t .

and hence $\omega = \omega(x, y) \rightarrow ①$

The equation of motion for an incompressible viscous fluid is

$$\frac{\partial \vec{q}}{\partial t} + (\vec{q} \cdot \vec{\nabla}) \vec{q} = \vec{F} - \frac{1}{\rho} \vec{\nabla} p + \nu \vec{\nabla}^2 \vec{q} \rightarrow ②$$

since the motion is steady and there is no body force acting on the fluid

$$\left. \begin{aligned} \frac{\partial \vec{q}}{\partial t} &= 0 \\ \vec{F} &= 0 \end{aligned} \right\} \rightarrow ③$$

By the use of ③, the equation ② becomes

$$(\vec{q} \cdot \vec{\nabla}) \vec{q} = -\frac{1}{\rho} \vec{\nabla} p + \nu \vec{\nabla}^2 \vec{q}$$

$$\begin{aligned} \Rightarrow \left(u \frac{\partial}{\partial x} + v \frac{\partial}{\partial y} + \omega \frac{\partial}{\partial z} \right) \omega \hat{k} &= -\frac{1}{\rho} \left[\frac{\partial p}{\partial x} \hat{i} + \frac{\partial p}{\partial y} \hat{j} + \frac{\partial p}{\partial z} \hat{k} \right] \\ &\quad + \nu \left[\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} + \frac{\partial^2}{\partial z^2} \right] \omega \hat{k} \end{aligned}$$

$$\Rightarrow \vec{0} = -\frac{1}{\rho} \left(\hat{i} \frac{\partial p}{\partial x} + \hat{j} \frac{\partial p}{\partial y} + \hat{k} \frac{\partial p}{\partial z} \right) + \nu \left[\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right] \omega \hat{k}$$

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$$\Rightarrow \hat{i} \frac{\partial \rho}{\partial x} + \hat{j} \frac{\partial \rho}{\partial y} + \hat{k} \frac{\partial \rho}{\partial z} = \mu \left(\frac{\partial \tilde{\omega}}{\partial x} + \frac{\partial \tilde{\omega}}{\partial y} \right) \hat{\omega}_k \quad \rightarrow (4)$$

By equating the coefficient of $\hat{i}, \hat{j}, \hat{k}$ in the equation (4)
we derive the following differential equation

$$\frac{\partial \rho}{\partial x} = 0 \quad \rightarrow (5.1)$$

$$\frac{\partial \rho}{\partial y} = 0 \quad \rightarrow (5.2)$$

$$\frac{\partial \rho}{\partial z} = \mu \left(\frac{\partial \tilde{\omega}}{\partial x} + \frac{\partial \tilde{\omega}}{\partial y} \right) \rightarrow (5.3)$$

equations (5.1) and (5.2) shows that ρ is independent of
 x and y .

further the motion is steady

$\therefore \rho$ is independent of t

$$\therefore \rho = \rho(z) \rightarrow (6)$$

By virtue of (6) the equation (5.3) becomes.

$$\mu \left(\frac{\partial \tilde{\omega}}{\partial x} + \frac{\partial \tilde{\omega}}{\partial y} \right) = \frac{d\rho}{dz}$$

\Rightarrow a function of x and y = a function of z .

Hence we must have

$$\mu \left(\frac{\partial \tilde{\omega}}{\partial x} + \frac{\partial \tilde{\omega}}{\partial y} \right) = \frac{d\rho}{dz} = \text{a constant} = -P \text{ (say)}$$

$$\Rightarrow \frac{\partial \tilde{\omega}}{\partial x} + \frac{\partial \tilde{\omega}}{\partial y} = -\frac{P}{\mu} \cdot \rightarrow (7)$$

2nd part

(89)

The equation of the cylinder is $\frac{x^2}{a^2} + \frac{y^2}{b^2} = 1$

The boundary condition for the flow problem is

$$\omega = 0, \text{ for } \frac{x^2}{a^2} + \frac{y^2}{b^2} = 1$$

$$\text{i.e. } \omega = 0 \text{ for } \frac{x^2}{a^2} + \frac{y^2}{b^2} - 1 = 0 \longrightarrow \textcircled{8}$$

In order to satisfy the condition $\textcircled{8}$, we take ω as follows,

$$\omega = A \left(1 - \frac{x^2}{a^2} - \frac{y^2}{b^2} \right) \longrightarrow \textcircled{9}$$

$$\therefore \textcircled{9} \Rightarrow \frac{\partial \omega}{\partial x} = A \left(-\frac{2x}{a^2} \right)$$

$$\frac{\partial \omega}{\partial x^2} = -\frac{2A}{a^2}$$

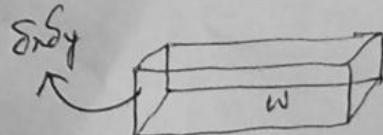
$$\text{II}^y, \quad \frac{\partial \omega}{\partial y^2} = -\frac{2A}{b^2}$$

$$\therefore \textcircled{7} \Rightarrow 2A \left(\frac{1}{a^2} + \frac{1}{b^2} \right) = \frac{P}{\mu}$$

$$\Rightarrow 2A \cdot \frac{a^2 + b^2}{a^2 b^2} = \frac{P}{\mu}$$

$$\Rightarrow A = \frac{P a^2 b^2}{2 \mu (a^2 + b^2)}$$

Let Q be the flux across any normal section of the cylinder



$$\therefore Q = \iint_R \omega dx dy, \text{ where } R = \left\{ (x, y) \mid \frac{x^2}{a^2} + \frac{y^2}{b^2} \leq 1 \right\} \quad \textcircled{10}$$

(90)

$$\therefore \textcircled{10} \Rightarrow Q = \iint_R A \left(1 - \frac{x^2}{a^2} - \frac{y^2}{b^2} \right) dx dy$$

$$= A \iint_R \left(1 - \frac{x^2}{a^2} - \frac{y^2}{b^2} \right) dx dy \longrightarrow \textcircled{11}$$

Consider the transformation $x = a r \cos \theta, y = b r \sin \theta$.

$$J = abr$$

$$\therefore \textcircled{11} \Rightarrow Q = A \int_{\theta=0}^{2\pi} \int_{r=0}^1 (1 - r^2) abr dr d\theta$$

$$= Aab \cdot 2\pi \int_{r=0}^1 (r - r^3) dr$$

$$= 2Aab\pi \left[\frac{r^2}{2} - \frac{r^4}{4} \right]_0^1$$

$$= 2\pi ab A \left(\frac{1}{2} - \frac{1}{4} \right)$$

$$= \frac{\pi ab A}{2}$$

$$= \frac{\pi ab}{2} \cdot \frac{P a^2 b^2}{2\mu(a^2 + b^2)}$$

$$\Rightarrow Q = \frac{P\pi a^2 b^2}{4\mu(a^2 + b^2)}$$

3rd part:

(91)

$$\text{We have } \omega = A \left(1 - \frac{x^v}{av} - \frac{y^v}{bv} \right)$$

$$= A \left[1 - \left(\frac{x^v}{av} + \frac{y^v}{bv} \right) \right] \longrightarrow 12$$

from 12 it is clear that ω is maximum when

$$\frac{x^v}{av} + \frac{y^v}{bv} = 0$$

i.e. when $x = 0, y = 0$.

And the maximum velocity is given by

$$\begin{aligned} \omega_{\max} &= A \\ &= \frac{P a^v b^v}{2 \mu (a^v + b^v)} \end{aligned}$$

#

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Ex. A liquid occupying the space between two co-axial circular cylinders is acted upon by a force $\frac{c}{r}$, per unit mass, where r is the distance from the axis, the line of force being circle around the axis. Prove that in the steady motion, the velocity at any point is given by

the formula

$$\frac{c}{2 \gamma} \left\{ \frac{b^v}{r} - \frac{r^v - a^v}{b^v - a^v} \log \frac{b}{a} - r \log \frac{r}{a} \right\}$$

where a and b are two radii and γ is the kinematic viscosity.

Soln Take the z -axis along the common axis of the two cylinders.

Let $\vec{q} = (q_r, q_\theta, q_z)$ be the fluid velocity at the point $P(r, \theta, z)$ in the fluid.

In the present case, since the lines of force are clearly round the z -axis, therefore $q_r = 0, q_z = 0$.

and let $q_\theta = \varphi$

$$\therefore \vec{q} = q_\theta \hat{\theta} = \varphi \hat{\theta}$$

The equation of continuity for an incompressible fluid is

$$\vec{\nabla} \cdot \vec{q} = 0$$

$$\Rightarrow \frac{1}{r \cdot r \cdot 1} \left[\frac{\partial}{\partial r} (r \cdot 1 \cdot q_r) + \frac{\partial}{\partial \theta} (1 \cdot 1 \cdot q_\theta) + \frac{\partial}{\partial z} (1 \cdot r \cdot q_z) \right] = 0$$

$$\Rightarrow \frac{\partial q_\theta}{\partial \theta} = 0$$

$$\Rightarrow \frac{\partial \varphi}{\partial \theta} = 0$$

$\therefore \varphi$ is independent of θ .

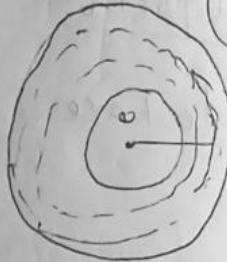
Again, since the motion is two dimensional with the z -axis planes as the plane of the motion.

Hence φ is independent of z

further, since the motion is steady

$\therefore \varphi$ is independent of t

$$\therefore \varphi = \varphi(r). \quad \rightarrow ①$$



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The equation of motion for an incompressible viscous fluid is (93)

$$\frac{\partial \vec{q}}{\partial t} + (\vec{q} \cdot \vec{\nabla}) \vec{q} = \vec{F} - \frac{1}{\rho} \vec{\nabla} p - 2 \vec{\nabla} \times \vec{\xi}$$

$$\Rightarrow \frac{\partial \vec{q}}{\partial t} + \frac{1}{2} \vec{\nabla} q^2 - \vec{q} \times \vec{\xi} = \vec{F} - \frac{1}{\rho} \vec{\nabla} p - 2 \vec{\nabla} \times \vec{\xi} \quad \rightarrow (4)$$

Since the motion is steady,

$$\frac{\partial \vec{q}}{\partial t} = 0 \quad \rightarrow (2)$$

\therefore force per unit mass

$$\vec{F} = \frac{c}{\rho} \hat{\theta} \quad \rightarrow (3)$$

$$\begin{aligned} \vec{\nabla} \vec{q}^2 &= \frac{\hat{r}}{r} \frac{\partial}{\partial r}(q^2) + \frac{\hat{\theta}}{r} \frac{\partial}{\partial \theta}(q^2) + \frac{\hat{z}}{r} \frac{\partial}{\partial z}(q^2) \\ &= r \cdot 2q_r' = 2q_r' \hat{r} \quad \rightarrow (4) \end{aligned}$$

$$\begin{aligned} \vec{\xi} &= \vec{\nabla} \times \vec{\xi} \\ &= \frac{1}{r \cdot r \cdot 1} \begin{vmatrix} \hat{r} & r\hat{\theta} & \hat{z} \\ \frac{\partial}{\partial r} & \frac{\partial}{\partial \theta} & \frac{\partial}{\partial z} \\ 0 & r\hat{q}_r & 0 \end{vmatrix} = \frac{\hat{z}}{r} \frac{d}{dr}(r\hat{q}_r) = \hat{z} \hat{q}_r' , \\ &\quad \vec{\xi} = \frac{1}{r} \frac{d}{dr}(r\hat{q}_r) . \end{aligned}$$

$$\vec{\nabla} \times \vec{\xi} = \frac{1}{r \cdot r \cdot 1} \begin{vmatrix} \hat{r} & r\hat{\theta} & \hat{z} \\ \frac{\partial}{\partial r} & \frac{\partial}{\partial \theta} & \frac{\partial}{\partial z} \\ 0 & 0 & \vec{\xi} \end{vmatrix} = -\hat{\theta} \frac{d\vec{\xi}}{dr} = -\hat{\theta} \vec{\xi}' \rightarrow (5)$$

$$\begin{aligned} \therefore \vec{q} \times \vec{\xi} &= q \hat{\theta} \times \hat{z} \hat{q}_r' \\ &= q \hat{z} \hat{q}_r' \quad \rightarrow (6) \end{aligned}$$

$$\therefore (4) \Rightarrow q_r' \hat{r} - q \hat{z} \hat{q}_r' = \frac{c}{\rho} \hat{\theta} - \frac{1}{\rho} \vec{\nabla} p + \nu \vec{\xi}' \hat{\theta}$$

$$\Rightarrow \varphi' \hat{\theta} - \varphi \hat{\theta}' = \frac{C}{r} \hat{\theta} - \frac{1}{\rho} \left[\hat{\gamma} \frac{\partial P}{\partial r} + \frac{\hat{\theta}}{r} \frac{\partial P}{\partial \theta} + \hat{\gamma} \frac{\partial P}{\partial \hat{\gamma}} \right] + 2 \hat{\xi}' \hat{\theta} \quad (7)$$

By equating the coefficient of $\hat{\theta}$, $\hat{\gamma}$ in the equation (7), we obtain the following differential equations.

$$\varphi' - \varphi \hat{\theta}' = - \frac{1}{\rho} \frac{\partial P}{\partial r} \rightarrow (8.1)$$

$$0 = \frac{C}{r} - \frac{1}{r\rho} \frac{\partial P}{\partial \theta} + 2 \hat{\xi}' \rightarrow (8.2)$$

$$0 = - \frac{1}{\rho} \frac{\partial P}{\partial \hat{\gamma}} \rightarrow (8.3)$$

(8.3) shows that P is independent of $\hat{\gamma}$.

Due to symmetry P is independent of θ .

Since, the motion is steady

$\therefore P$ is independent of ' t '

$$\therefore P = P(r) \rightarrow (9)$$

By virtue of (9), the equation (8.2) reduces to

$$\hat{\xi}' = - \frac{C}{r}$$

$$\Rightarrow \hat{\xi}' = - \frac{C}{r^2}$$

$$\Rightarrow d\hat{\xi} = - \frac{C}{r^2} dr$$

$$\Rightarrow \hat{\xi} = - \frac{C}{r} \log r + A$$

$$\Rightarrow \frac{d}{dr} (\varphi \hat{\xi}) = A - \frac{C}{r} \log r$$

$$\Rightarrow \frac{d}{dr} (\varphi \hat{\xi}) = Ar - \frac{C}{r} \log r$$

$$\Rightarrow \varphi \hat{\xi} = \frac{Ar^2}{2} - \frac{C}{r} \left[\log r \cdot \frac{r}{2} - \int \frac{1}{r} \cdot \frac{r}{2} dr \right] + B$$

$$\Rightarrow q = \frac{Ar^{\nu}}{2} - \frac{C}{\nu} \left[\frac{r^{\nu}}{2} \log r - \frac{r^{\nu}}{4} \right] + B$$

(95)

$$\Rightarrow q = \frac{Ar}{2} - \frac{Cr^{\nu}}{2\nu} \left(\log r - \frac{1}{2} \right) + \frac{B}{r}$$

$$= \frac{Ar}{2} - \frac{Cr^{\nu}}{2\nu} \log r + \frac{Cr^{\nu}}{2\nu} + \frac{B}{r}$$

$$= Kr - \frac{Cr^{\nu}}{2\nu} \log r + \frac{B}{r} \quad \rightarrow (1), \quad K = \frac{A}{2} + \frac{C}{4\nu}$$

The boundary condition for the slow problem are,

$$q = 0, \text{ at } r = a \quad \rightarrow (1.1)$$

$$q = 0, \text{ at } r = b \quad \rightarrow (1.2)$$

(5) can be written as

$$\frac{q}{r} = K - \frac{C}{2\nu} \log r + \frac{B}{r^{\nu}} \quad \rightarrow (12)$$

subjecting (12) to the condition (1.1) and (1.2), we get,

$$0 = K - \frac{C}{2\nu} \log a + \frac{B}{a^{\nu}} \quad \rightarrow (13.1)$$

$$0 = K - \frac{C}{2\nu} \log b + \frac{B}{b^{\nu}} \quad \rightarrow (13.2)$$

$$\therefore (13.1) - (13.2) \Rightarrow$$

$$0 = \frac{C}{2\nu} \log \frac{b}{a} + B \left(\frac{1}{a^{\nu}} - \frac{1}{b^{\nu}} \right)$$

$$\Rightarrow 0 = \frac{C}{2\nu} \log \frac{b}{a} + B \cdot \frac{b^{\nu} - a^{\nu}}{a^{\nu} b^{\nu}}$$

$$\Rightarrow B \cdot \frac{b^{\nu} - a^{\nu}}{a^{\nu} b^{\nu}} = - \frac{C}{2\nu} \log \frac{b}{a}$$

$$\Rightarrow B = - \frac{C a^{\nu} b^{\nu} \log \frac{b}{a}}{2\nu (b^{\nu} - a^{\nu})}$$

(96)

(12) - (3.1) \Rightarrow

$$\frac{q}{r} = \frac{c}{2r} \log \frac{q}{r} + B \left(\frac{1}{r^2} - \frac{1}{q^2} \right)$$

$$= \frac{c}{2r} \log \frac{q}{r} - \frac{c a^v b^v \log \frac{b}{a}}{2r (b^v - a^v)} \cdot \frac{a^v - r^v}{a^v r^v}$$

$$= \frac{c}{2r} \left[\log \frac{q}{r} - \frac{b^v (a^v - r^v)}{r^v (b^v - a^v)} \log \frac{b}{a} \right]$$

$$= \frac{c}{2r} \left[\frac{(b^v - a^v) b^v - \log \frac{b}{a}}{r^v (b^v - a^v)} - \log \frac{b}{a} \right]$$

$$\Rightarrow q = \frac{c}{2r} \left[\frac{(b^v - a^v) b^v - \log \frac{b}{a}}{r (b^v - a^v)} - r \log \frac{b}{a} \right] \#$$

Ex. The space between two coaxial cylinders of radii a and b is filled with a viscous fluid and the cylinders are made to rotate with angular velocities ω_1 and ω_2 . Prove that in the steady motion the angular velocity of the fluid is given by

$$\omega = \frac{a^v (b^v - r^v) \omega_1 + b^v (r^v - a^v) \omega_2}{r^v (b^v - a^v)}.$$

Soln: Proceeding as in the previous example and putting $c = 0$, we get $\xi' = 0$

$$\Rightarrow \frac{d\xi}{dr} = 0$$

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$$\Rightarrow \xi = A = \text{a constant}$$

$$\Rightarrow \frac{1}{r} \frac{d}{dr}(r\varphi) = A$$

$$\Rightarrow \frac{d}{dr}(r\varphi) = Ar$$

$$\Rightarrow r\varphi = A \frac{r^2}{2} + B$$

$$\Rightarrow \varphi = \frac{Ar}{2} + \frac{B}{r}$$

$$\Rightarrow r\omega = \frac{Ar}{2} + \frac{B}{r}$$

$$\Rightarrow \omega = \frac{A}{2} + \frac{B}{r^2}$$

$$\Rightarrow \omega = \lambda + \frac{B}{r^2} \rightarrow \textcircled{A}, \lambda = \frac{A}{2}$$

The boundary conditions for the flow problem are

$$\omega = \omega_1 \text{ at } r=a \rightarrow \textcircled{1.1}$$

$$\omega = \omega_2 \text{ at } r=b \rightarrow \textcircled{1.2}$$

Substituting \textcircled{A} to the condition \textcircled{1.1} & \textcircled{1.2}, we get,

$$\omega_1 = \lambda + \frac{B}{a^2} \rightarrow \textcircled{2.1}$$

$$\omega_2 = \lambda + \frac{B}{b^2} \rightarrow \textcircled{2.2}$$

$$\begin{aligned} \textcircled{2.1} - \textcircled{2.2} \Rightarrow \omega_1 - \omega_2 &= B \left(\frac{1}{a^2} - \frac{1}{b^2} \right) \\ &= B \cdot \frac{b^2 - a^2}{a^2 b^2} \end{aligned}$$

$$\Rightarrow B = \frac{a^2 b^2 (\omega_1 - \omega_2)}{b^2 - a^2}$$

$$\textcircled{A} - \textcircled{2.1} \Rightarrow \omega - \omega_1 = B \left(\frac{1}{r^2} - \frac{1}{a^2} \right)$$

$$= \frac{a^2 b^2 (\omega_1 - \omega_2)}{(b^2 - a^2)} \left(\frac{a^2 - r^2}{a^2 b^2} \right)$$

(98)

$$\Rightarrow \omega = \omega_1 + \frac{b^v(a^v - rr)}{r^v(b^v - av)} (\omega_1 - \omega_2)$$

$$= \frac{\omega_1 r^v(b^v - av) + b^v(a^v - rr)\omega_1 - b^v(a^v - rr)\omega_2}{r^v(b^v - av)}$$

$$= \frac{\omega_1 (b^v r^v - a^v r^v + b^v a^v - b^v a^v) + b^v(r^v - av)\omega_2}{r^v(b^v - av)}$$

$$\Rightarrow \omega = \frac{\omega_1 a^v(b^v - rr) + \omega_2 (r^v - av) b^v}{r^v(b^v - av)}$$

#

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Ex. Two concentric infinite circular cylinders of radii a, b ($b > a$) contains a viscous liquid in between the liquid is set in motion by relatively the other cylinder with a constant angular velocity ω , write the inner cylinder is kept at rest. Discuss the motion and show that the couple per unit length on the inner cylinder due to the viscous drag has moment $\frac{4\pi\mu a^v b^v \omega}{b^v - av}$.

Solⁿ Proceeding as previous example to the state

$$\omega = \lambda + \frac{\beta}{r^v} \longrightarrow (A)$$

The boundary conditions for the present flow problem are $\omega = \omega$ for $r = b \longrightarrow (I)$ $\omega = 0$ for $r = a \longrightarrow (II)$

Subjecting A) to ① and ⑪, we get,

(99)

$$-\omega = \lambda + \frac{B}{b^2} \longrightarrow (2.1)$$

$$\theta = \lambda + \frac{B}{a^2} \longrightarrow (2.2)$$

$$\therefore (2.1) - (2.2) \Rightarrow -\omega = B \left(\frac{1}{a^2} - \frac{1}{b^2} \right)$$

$$= B \frac{b^2 - a^2}{a^2 b^2}$$

$$\Rightarrow B = \frac{a^2 b^2 \omega}{a^2 - b^2}$$

$$A) - (2.2) \Rightarrow$$

$$\omega = B \left(\frac{1}{b^2} - \frac{1}{a^2} \right)$$

$$= \frac{a^2 b^2 \omega}{a^2 - b^2} \left(\frac{a^2 - b^2}{a^2 b^2} \right)$$

$$= \frac{b^2 (a^2 - b^2) \omega}{\pi^2 (a^2 - b^2)}$$

The moment of couple present length of the inner cylinder due to viscous that is given by

$$G = \left[(\tau_{ro} \cdot 2\pi r \cdot l)^r \right]_{r=a}$$

$$= \left[2\pi r^2 \tau_{ro} \right]_{r=a} \longrightarrow ③$$

We have,

$$\tau_{ro} = \mu \left[\frac{\partial \varphi}{\partial r} - \frac{\varphi}{r} \right]$$

(150)

$$= \mu \left[\frac{\partial}{\partial r}(\gamma\omega) - \frac{\gamma\omega}{r} \right]$$

$$= \mu \left[\gamma \frac{\partial \omega}{\partial r} + \omega - \omega \right]$$

$$= \mu \gamma \frac{\partial \omega}{\partial r}$$

$$= \mu \gamma \cdot \frac{b^{\nu}-r^{\nu}}{a^{\nu}-b^{\nu}} \frac{d}{dr} \left(\frac{a^{\nu}-r^{\nu}}{r^{\nu}} \right)$$

$$= \mu \gamma \cdot \frac{b^{\nu}-r^{\nu}}{a^{\nu}-b^{\nu}} \frac{d}{dr} \left(\frac{a^{\nu}}{r^{\nu}} - 1 \right)$$

$$= \frac{\mu \gamma b^{\nu}-r^{\nu}}{a^{\nu}-b^{\nu}} \left[a^{\nu}(-2) r^{-3} \right]$$

$$= \frac{\mu \gamma b^{\nu}-r^{\nu}}{b^{\nu}-a^{\nu}} \frac{2a^{\nu}}{r^3}$$

$$\Rightarrow h = 2\pi r^{\nu} \frac{2a^{\nu}b^{\nu}\mu-2}{(b^{\nu}-a^{\nu})r^{\nu}} \Big|_{r=a}$$

$$= \frac{4\pi a^{\nu}b^{\nu}\mu-2}{(b^{\nu}-a^{\nu})} \quad \#$$